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Advanced Automatic Control MDP 444

If you have a smart project, you can say "I'm an engineer"

Lecture 4

Staff boarder

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Advanced Automatic Control MDP 444

- Lecture aims:
 - Understand the Block reduction techniques
 - Identify the transfer function
 - Be aware by modling multiple technique

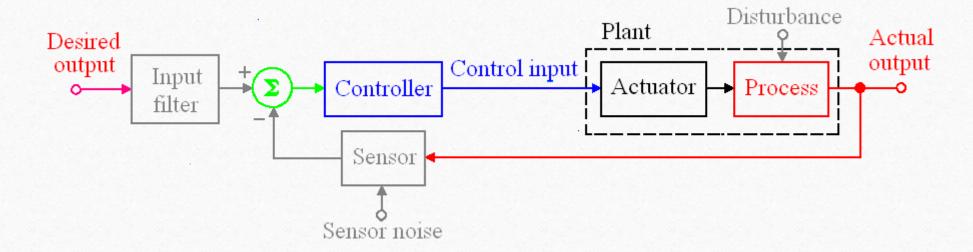
Mathematical Modeling

Transfer Function
 Transfer Function
 Block Diagram

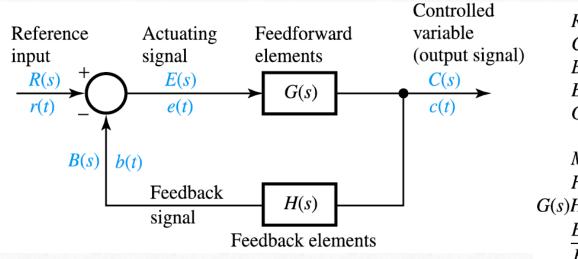
Block Diagram

Signal Flow

Component Block Diagram



Component Block Diagram



- R(s) Reference input
- C(s) Output signal (controlled variable)
- B(s) Feedback signal = H(s)C(s)
- E(s) Actuating signal (error) = [R(s) B(s)]
- G(s) Forward path transfer function or open-loop transfer function = C(s)/E(s)
- M(s) Closed-loop transfer function = C(s)/R(s) = G(s)/[1 + G(s)H(s)]
- H(s) Feedback path transfer function
- G(s)H(s) Loop gain
 - $\frac{E(s)}{R(s)}$ = Error-response transfer function $\frac{1}{1 + G(s)H(s)}$

TABLE 3.4.1 Some of the Block Diagram Reduction Manipulations

Original Plank Diagram	Manimulation	Madified Block Diagram
Original Block Diagram	Manipulation	Modified Block Diagram
$R \longrightarrow G_1 \longrightarrow G_2 \longrightarrow$	Cascaded elements	$R \longrightarrow G_1G_2 \longrightarrow$
$R \longrightarrow G_1 \longrightarrow C_2 \longrightarrow C_2$	Addition or subtraction (eliminating auxiliary forward path)	$\xrightarrow{R} G_1 \pm G_2 \xrightarrow{C}$
$R \longrightarrow G \longrightarrow C \longrightarrow$	Shifting of pickoff point ahead of block	$ \begin{array}{c} R \\ \hline G \end{array} $
$R \longrightarrow G \longrightarrow C$	Shifting of pickoff point behind block	$ \begin{array}{cccccccccccccccccccccccccccccccccccc$
$ \begin{array}{c} R \longrightarrow G \longrightarrow C \\ & \downarrow C \end{array} $	Shifting summing point ahead of block	$ \begin{array}{cccccccccccccccccccccccccccccccccccc$
$ \begin{array}{c} R & \stackrel{+}{\longrightarrow} C \\ & \stackrel{-}{\longrightarrow} C \end{array} $	Shifting summing point behind block	$\stackrel{R}{\longrightarrow} G \stackrel{+}{\longrightarrow} \stackrel{E}{\longrightarrow} C$
$ \begin{array}{cccccccccccccccccccccccccccccccccccc$	Removing <i>H</i> from feedback path	$\stackrel{R}{\longrightarrow} 1/H \stackrel{+}{\longrightarrow} H \stackrel{G}{\longrightarrow} C$
$\begin{array}{c} R \xrightarrow{+} G & C \\ & & \\ & $	Eliminating feedback path	$\xrightarrow{R} \qquad \xrightarrow{G} \qquad \xrightarrow{C}$

A **signal-flow graph** is a diagram consisting of nodes that are connected by several directed branches and is a graphical representation of a set of linear relations.

The basic element of a signal-flow graph is a unidirectional path segment called a **branch**

A **loop** is a closed path that originates and terminates on the same node. Two loops are said to be **nontouching** if they do not have a common node

$$T_{ij} = \frac{\sum_{k} P_{ijk} \, \Delta_{ijk}}{\Delta},$$

 $P_{ijk} = \text{gain of } k \text{th path from variable } x_i \text{ to variable } x_j,$

 Δ = determinant of the graph,

 $\Delta_{ijk} = \text{cofactor of the path } P_{ijk}$,

$$\Delta = 1 - \sum_{n=1}^{N} L_n + \sum_{\substack{n,m \text{nontouching}}} L_n L_m - \sum_{\substack{n,m,p \text{nontouching}}} L_n L_m L_p + \cdots \Delta = 1 - (\text{sum of all different loop gains}) + (\text{sum of the gain products of all combinations of two nontouching loops}) - (\text{sum of the gain products of all combinations of three nontouching loops}) + \cdots$$

The cofactor Δ_{iik} is the determinant with the loops touching the kth path removed.

The paths connecting the input R(s) and output Y(s) are

$$P_1 = G_1G_2G_3G_4$$
 (path 1) and $P_2 = G_5G_6G_7G_8$ (path 2)

There are four self-loops:

$$L_1 = G_2H_2$$
, $L_2 = H_3G_3$, $L_3 = G_6H_6$, and $L_4 = G_7H_7$

Loops L1 and L2 do not touch L3 and L4. Therefore, the determinant is R(s)

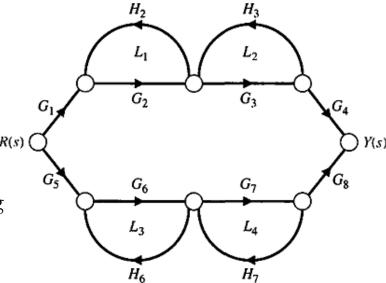
$$\Delta = 1 - (L_1 + L_2 + L_3 + L_4) + (L_1L_3 + L_1L_4 + L_2L_3 + L_2L_4)$$

The cofactor of the determinant along path 1 is evaluated by removing the loops that touch path 1 from Δ . $\Delta_1 = 1 - (L_3 + L_4)$

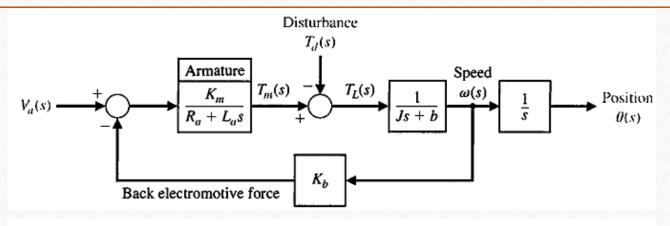
Similarly, the cofactor for path 2 is $\Delta_2 = 1 - (L_1 + L_2)$

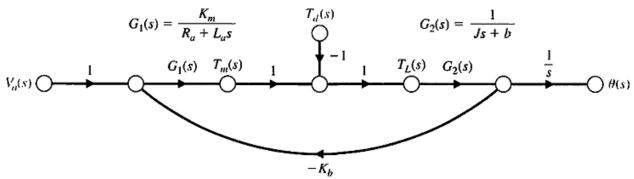
Therefore, the transfer function of the system is

$$\frac{Y(s)}{R(s)} = T(s) = \frac{P_1 \Delta_1 + P_2 \Delta_2}{\Delta} = \frac{G_1 G_2 G_3 G_4 (1 - L_3 - L_4) + G_5 G_6 G_7 G_8 (1 - L_1 - L_2)}{1 - L_1 - L_2 - L_3 - L_4 + L_1 L_3 + L_1 L_4 + L_2 L_3 + L_2 L_4}$$



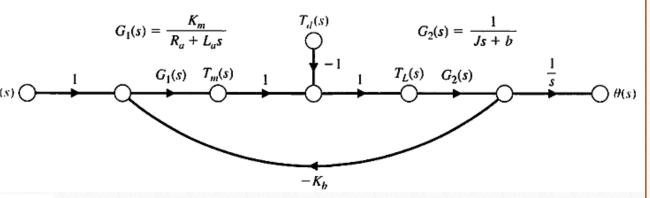
The armature-controlled DC motor





The armature-controlled DC motor

Using Mason's signal-flow gain formula, transfer function for $\theta(s)/Va(s)$ with $Td(s) = Q_{n(s)}$ The forward path is P1(s), which touches the one loop, L1(s), where



$$P_1(s) = \frac{1}{s}G_1(s)G_2(s) \text{ and } L_1(s) = -K_bG_1(s)G_2(s).$$

$$\Delta = 1 - (L_1 + L_2 + L_3 + L_4) + (L_1L_3 + L_1L_4 + L_2L_3 + L_2L_4)$$

$$T(s) = \frac{P_1(s)}{1 - L_1(s)} = \frac{(1/s)G_1(s)G_2(s)}{1 + K_bG_1(s)G_2(s)} = \frac{K_m}{s[(R_a + L_as)(Js + b) + K_bK_m]},$$

State Space Equations

• **State equations** is a description which relates the following four elements: input, system, state variables, and output

$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}\mathbf{u}(t)$$

$$\mathbf{y}(t) = \mathbf{C}\mathbf{x}(t) + \mathbf{D}\mathbf{u}(t)$$

Matrix A has dimensions nxn and it is called the **system** matrix, having the general form

$$\mathbf{A} = \begin{bmatrix} a_{11} & a_{12} & \cdots & a_{1n} \\ a_{21} & a_{22} & \cdots & a_{2n} \\ \vdots & \vdots & & \vdots \\ a_{n1} & a_{n2} & \cdots & a_{nn} \end{bmatrix}$$

Matrix B has dimensions nxm and it is called the **input** matrix, having the general

form

Matrix C has dimensions pxn and it is called the **output** matrix, having the general form

$$\mathbf{C} = \begin{bmatrix} c_{11} & c_{12} & \cdots & c_{1n} \\ c_{21} & c_{22} & \cdots & c_{2n} \\ \vdots & \vdots & & \vdots \\ c_{p1} & c_{p2} & \cdots & c_{pn} \end{bmatrix}$$

Matrix D has dimensions pxm and it is called the **feedforward** matrix, having the general form

$$\mathbf{B} = \begin{bmatrix} b_{11} & b_{12} & \cdots & b_{1m} \\ b_{21} & b_{22} & \cdots & b_{2m} \\ \vdots & \vdots & & \vdots \\ b_{n1} & b_{n2} & \cdots & b_{nm} \end{bmatrix}$$

$$\mathbf{D} = \begin{bmatrix} d_{11} & d_{12} & \cdots & d_{1m} \\ d_{21} & d_{22} & \cdots & d_{2m} \\ \vdots & \vdots & & \vdots \\ d_{p1} & d_{p2} & \cdots & d_{pm} \end{bmatrix}$$

State Space

The general form of a dynamic system

The concept of a set of state variables that represent a dynamic system can be illustrated in terms of the spring-mass-damper system. A set of state variables sufficient to describe this system includes the position and the velocity of the mass.

• We will define a set of state variables as (x1, x2), where

$$x_1(t) = y(t)$$
 and $x_2(t) = \frac{dy(t)}{dt}$. $\frac{dx_1}{dt} = x_2$

To write Equation of motion in terms of the state variables, we substitute the state variables as already defined and obtain

$$M\frac{dx_2}{dt} + bx_2 + kx_1 = u(t)$$

$$M\frac{d^2y}{dt^2} + b\frac{dy}{dt} + ky = u(t)$$

Wall

Therefore, we can write the equations that describe the behavior of the spring-mass damper system as the set of two first-order differential equations $\frac{1}{4r_0} = \frac{1}{r_0}$

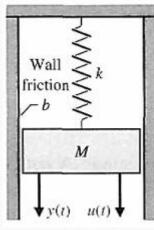
$$\frac{dx_2}{dt} = \frac{-b}{M}x_2 - \frac{k}{M}x_1 + \frac{1}{M}u$$

State Space

State space matrix

$$\frac{dx_1}{dt} = x_2$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -k & -b \\ \hline m & m \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \\ \hline m \end{bmatrix} \begin{bmatrix} u_{(t)} \end{bmatrix}$$



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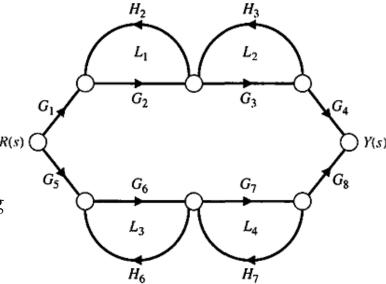
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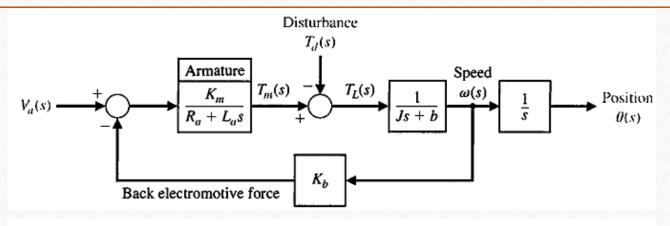
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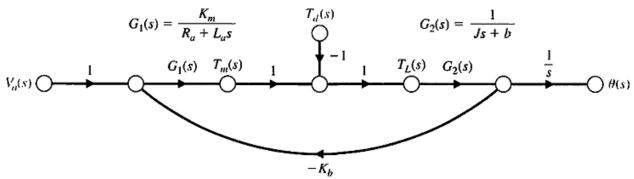
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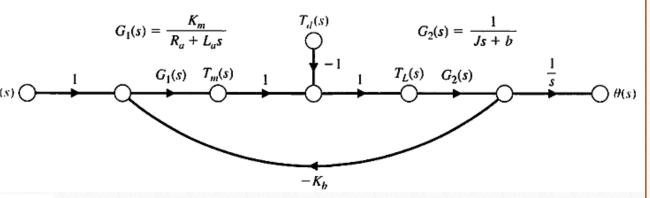
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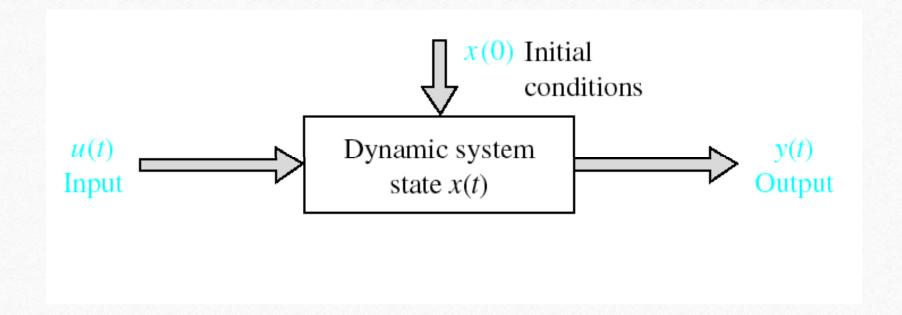
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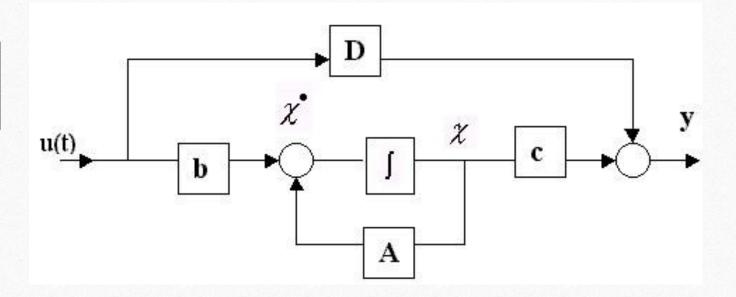
The general form of a dynamic system



State Space Equations

$$SISO \Rightarrow \dot{X} = Ax(t) + Bu(t)$$

 $Y = Cx(t) + Du(t)$



The general form of a dynamic system

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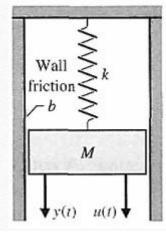
Therefore, we can write the equations that describe the behavior of the spring-mass damper system as the set of two first-order differential equations $\frac{1}{4x^2} = \frac{1}{4x^2}$

$$\frac{dx_2}{dt} = \frac{-b}{M}x_2 - \frac{k}{M}x_1 + \frac{1}{M}u$$

State space matrix

$$\frac{dx_2}{dt} = \frac{-b}{M}x_2 - \frac{k}{M}x_1 + \frac{1}{M}u$$

$$\begin{bmatrix} \dot{x_1} \\ \dot{x_2} \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -k & -b \\ m \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \\ m \end{bmatrix} [u_{(t)}]$$



$$M\frac{d^2y}{dt^2} + b\frac{dy}{dt} + ky = u(t)$$

RLC circuit example

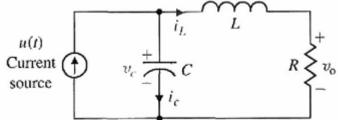
- The state of this system can be described by a set of state variables (x1, x2), where x1 is the capacitor voltage vo(t) and x2 is the inductor current $i_L\{t\}$.
- Utilizing Kirchhoff's current law at the junction

$$i_c = C\frac{dv_c}{dt} = +u(t) - i_L$$

Kirchhoff's voltage law for the right-hand loop provides the equation describing the rate of change of inductor current as $L\frac{di_L}{dt} = -Ri_L + v_c$

The output of this system is represented

$$v_{\rm o} = Ri_L(t)$$



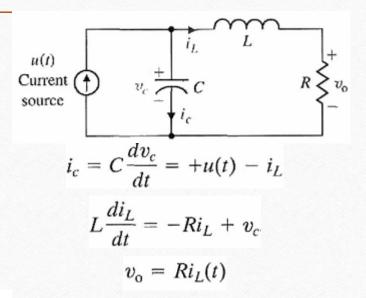
RLC circuit example

• rewrite Equations as a set of two first-order differential equations in terms of the state variables x1 and x2 as follows: dx_1 1 1 dx_2 1 dx_3 1 dx_4 1 dx_2 1 dx_3

$$\frac{dx_1}{dt} = -\frac{1}{C}x_2 + \frac{1}{C}u(t) \qquad \frac{dx_2}{dt} = +\frac{1}{L}x_1 - \frac{R}{L}x_2$$

- The output signal is then $y_1(t) = v_0(t) = Rx_2$
- obtain the state variable differential equation for the RLC
- and the output as $y = [0 \ R]x$

$$\dot{\mathbf{x}} = \begin{bmatrix} 0 & \frac{-1}{C} \\ \frac{1}{L} & \frac{-R}{L} \end{bmatrix} \mathbf{x} + \begin{bmatrix} \frac{1}{C} \\ 0 \end{bmatrix} u(t)$$



TRANSFER FUNCTION FROM THE STATE EQUATION

• Obtain a transfer function G(s), Given the state variable equations. Recalling Equations :where v is the single output and u is the single input. $\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}u$ $\mathbf{y} = \mathbf{C}\mathbf{x} + \mathbf{D}u$

The Laplace transforms of Equations sX(s) = AX(s) + BU(s)Y(s) = CX(s) + DU(s)

where B is an $n \times 1$ matrix, since u is a single input, we obtain

$$(s\mathbf{I} - \mathbf{A})\mathbf{X}(s) = \mathbf{B}U(s)$$

 $\mathbf{X}(s) = \Phi(s)\mathbf{B}U(s)$

• we obtain state transition Matrix

$$[s\mathbf{I} - \mathbf{A}]^{-1} = \mathbf{\Phi}(s)$$

TRANSFER FUNCTION FROM THE STATE EQUATION

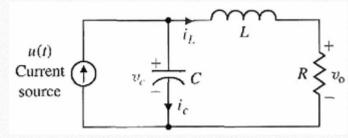
• Transfer function G(s): G(s) = Y(s)/U(s) is

$$G(s) = \mathbf{C}\Phi(s)\mathbf{B} + \mathbf{D}$$

• Let us determine the transfer function G(s) = Y(s)/U(s) for the RLC circuit, described by the differential equations

$$\dot{\mathbf{x}} = \begin{bmatrix} 0 & \frac{-1}{C} \\ \frac{1}{L} & \frac{-R}{L} \end{bmatrix} \mathbf{x} + \begin{bmatrix} \frac{1}{C} \\ 0 \end{bmatrix} u$$

$$y = \begin{bmatrix} 0 & R \end{bmatrix} \mathbf{x}.$$



TRANSFER FUNCTION FROM THE STATE EQUATION

• Then we have

$$[s\mathbf{I} - \mathbf{A}] = \begin{bmatrix} s & \frac{1}{C} \\ \frac{-1}{L} & s + \frac{R}{L} \end{bmatrix}$$

• Therefore, we obtain

$$\Phi(s) = [s\mathbf{I} - \mathbf{A}]^{-1} = \frac{1}{\Delta(s)} \begin{bmatrix} \left(s + \frac{R}{L}\right) & \frac{-1}{C} \\ \frac{1}{L} & s \end{bmatrix} \Delta(s) = s^2 + \frac{R}{L}s + \frac{1}{LC}$$

• Then the transfer function is

$$G(s) = \begin{bmatrix} 0 & R \end{bmatrix} \begin{bmatrix} \frac{s + \frac{R}{L}}{\Delta(s)} & \frac{-1}{C\Delta(s)} \\ \frac{1}{L\Delta(s)} & \frac{s}{\Delta(s)} \end{bmatrix} \begin{bmatrix} \frac{1}{C} \\ 0 \end{bmatrix} = \frac{R/(LC)}{\Delta(s)} = \frac{R/(LC)}{s^2 + \frac{R}{L}s + \frac{1}{LC}}$$

• Transfer from time domain to frequency domain:

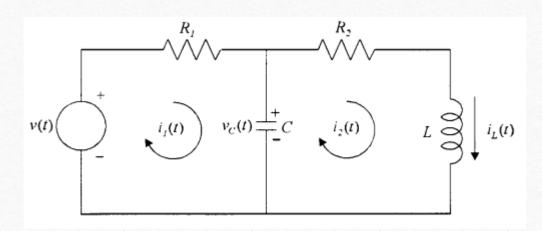
$$R_1 i_1(t) + \frac{1}{C} \int_0^t i_1(t) dt - \frac{1}{C} \int_0^t i_2(t) dt = v(t)$$

$$\left[R_1 + \frac{1}{Cs} \right] I_1(s) - \frac{1}{Cs} I_2(s) = V(s)$$

$$-\frac{1}{C} \int_0^t i_1(t) dt + R_2 i_2(t) + L \frac{di_2}{dt} + \frac{1}{C} \int_0^t i_2(t) dt = 0$$
$$-\frac{1}{Cs} I_1(s) + \left[R_2 + Ls + \frac{1}{Cs} \right] I_2(s) = 0$$

• Transfer function

$$\frac{I_2(s)}{V(s)} = \frac{Cs}{(R_1Cs+1)(LCs^2+R_2Cs+1)-1} = \frac{1}{R_1LCs^2+(R_1R_2C+L)s+R_1+R_2}$$

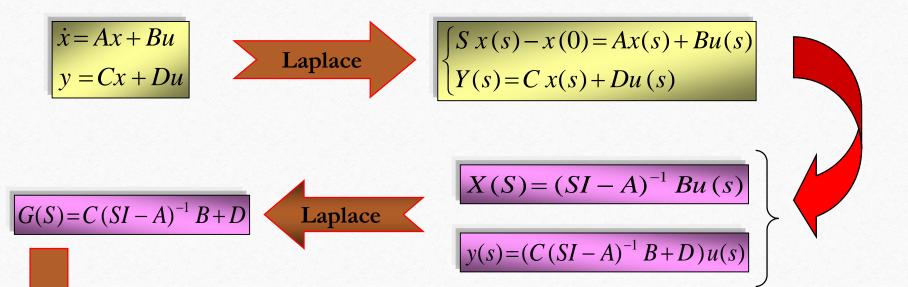


$$\begin{cases} e(t) - R_1 i_1(t) - L_1 \frac{di_1}{dt} - V_C(t) = \phi \\ V_C(t) - L_2 \frac{di_2}{dt} - R_2 i_2 = \phi \\ i_c = i_1 - i_2 = C \frac{dv_c}{dt} \end{cases}$$

$$x = (i_1 \ i_2 \ v_c)^T$$

$$X^{\bullet} = \begin{pmatrix} \frac{-R_1}{L_1} & 0 & \frac{-1}{L_1} \\ 0 & \frac{-R_2}{L_2} & \frac{1}{L_2} \\ \frac{1}{C} & \frac{-1}{C} & 0 \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \\ x_3 \end{pmatrix} + \begin{pmatrix} \frac{1}{L_1} \\ 0 \\ 0 \end{pmatrix} e(t)$$

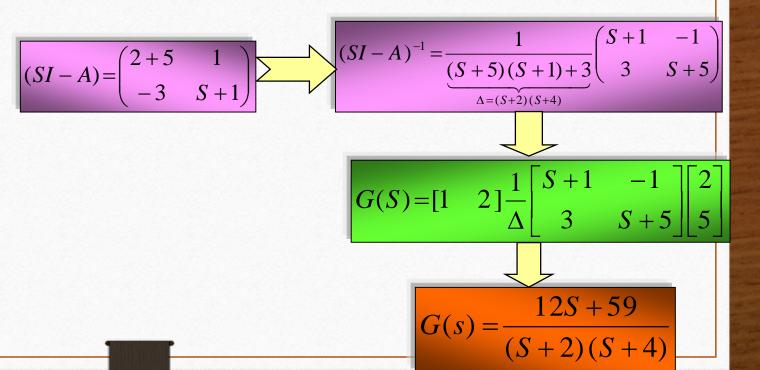
$$y(t) = (0 \quad R_2 \quad 0) \quad \begin{pmatrix} x_1 \\ x_2 \\ x_3 \end{pmatrix}$$



$$\begin{cases} \dot{x}_1 = -5x_1 - x_2 + 2u \\ \dot{x}_2 = 3x_1 - x_2 + 5u \end{cases} \Rightarrow \dot{x} = \begin{pmatrix} -5 & -1 \\ 3 & -1 \end{pmatrix} x + \begin{pmatrix} 2 \\ 5 \end{pmatrix} u$$

$$y = x_1 + 2x_2$$

$$y = (1 \quad 2) x$$



Model Examples

Pulse Width Modulation (PWM)

